

Mass

7530-286-6952

FEDERAL SUPPLY SERVICE

GPO : 1964-O-727-185

Measurement of mass under zero G conditions -
Most of the methods proposed so far depend
upon 1) making the mass part of an oscillatory
system by the addition of a compliance and
measuring periods: $\frac{1}{T} \approx \frac{1}{\sqrt{2\pi MC}}$

2) ~~spinning it~~ placing the mass in a circular
orbit and measuring the force developed.

$$F = \frac{1/2 m v^2}{R}$$

3)

The difficulties in method 1) differ from lack of complete coupling of all parts of a man's body.

2) variations in distribution of mass as a function of length during measurement

Reference to 2.1 -

Coermans et al. Luftfahrtforsch. V 3 P 111
1938 - see also V-III P-6636 668 Medical Physics
show that deviation from linearity ($m\omega$) is negligible
below 3 CPS when sitting in abdominal restraints.

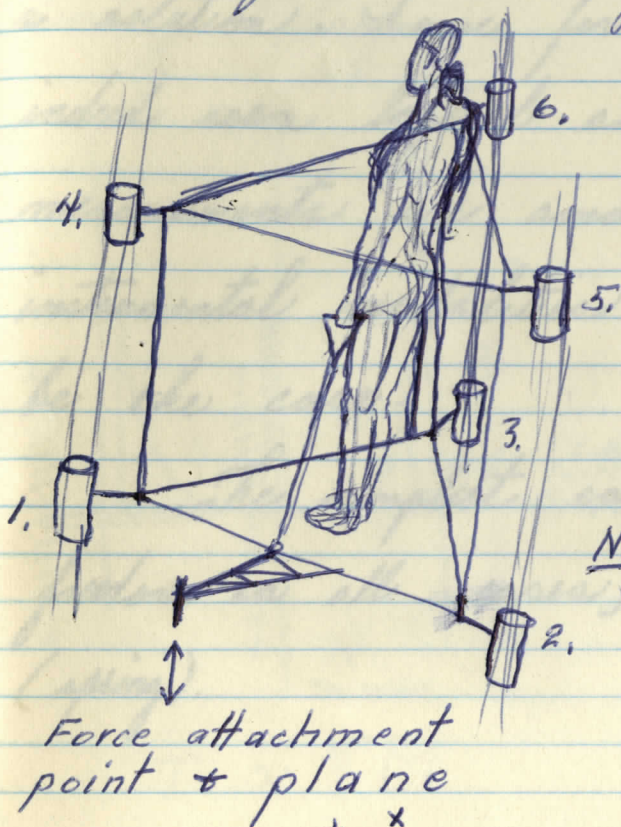
Fig. 1 { $\theta = \frac{y}{r}$ + $r = \frac{y}{\theta}$

R = Force applied Y = resulting Force in Y plane

$1^\circ = .9998$ $2^\circ 30' = .999$ (.1%)

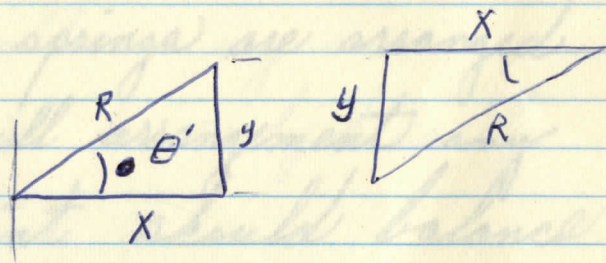
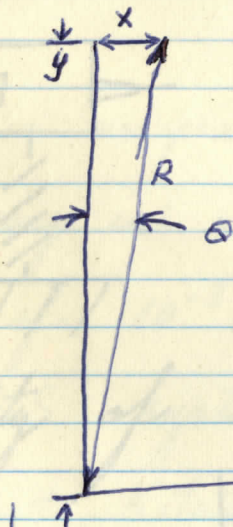
It would seem hopeless to attempt to solve the problem of restraining the mass to a single degree of freedom. In addition a reduction in the motion of individual body components may be effected -

1-6 are low friction bearings to restrain the platform to a single degree of freedom - (Vertical)



Not to SCALE or Proportion

Variations of force as a function of variation from a given plane



$$F = R - y$$

$$R = \frac{\tan \theta}{x}$$

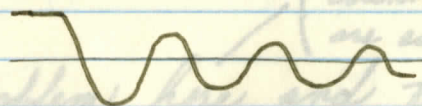
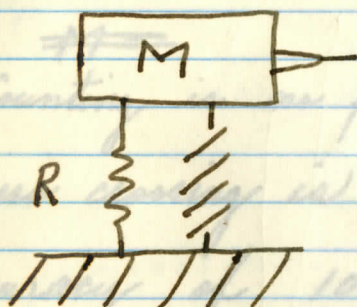
$$\text{Fig. 1} \left\{ \begin{array}{l} \cos \theta = \frac{y}{R} \\ y = \frac{R \cos \theta}{\cos \theta} \end{array} \right.$$

R = Force applied y = resulting Force in Y plane

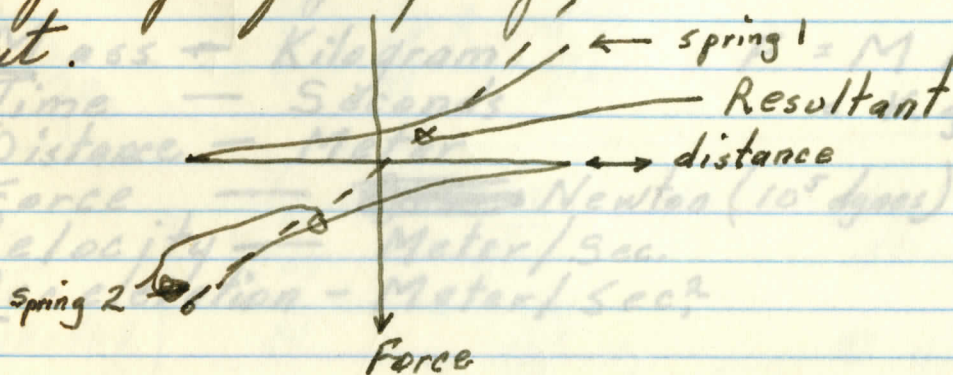
$$1^\circ = .9998 \quad 2^\circ 30' = .999 \quad (.1\%)$$

At first ~~of~~ glance an analog of capacitor R.C.R. circuit measurements might seem most logical for a solution. Some form of frequency determination would indeed seem logical since time & frequency measurements are among our most accurate instrumental modalities. This however may not be the case -

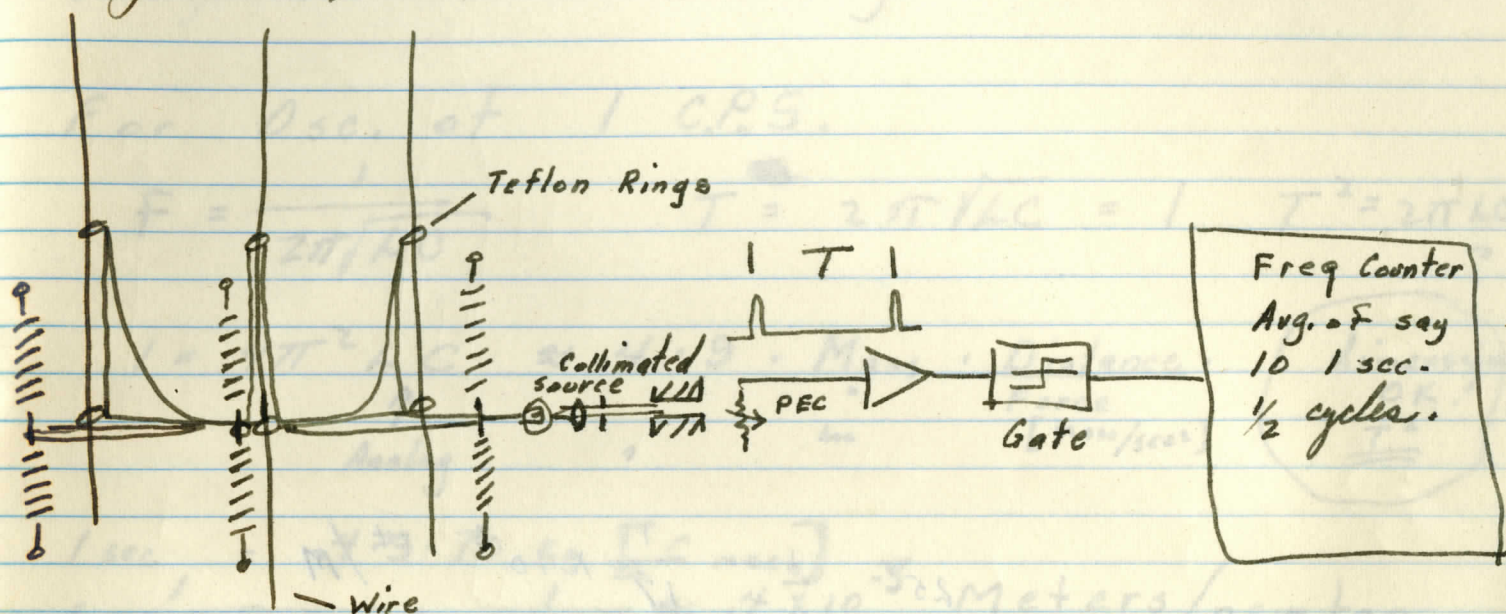
The simplest case is (assuming 1 degree of freedom in all cases) addition of a compliance (spring).



If the springs are arranged in a push-pull arrangement any irregularity of spring constant should balance out.



A displacement could be applied and the frequency of the resultant damped oscillation measured by timing techniques.



$$T \sim \sqrt{M}$$

~~14~~ (counting accuracies of 5×10^{-5} are easily available)
Counting is no problem here and the detection of zero crossing is a mechanical problem - say an accuracy of 10^{-3} ins. and should be realized easily.

In M K S system -

Mass - Kilogram

Time - Seconds

Distance - Meter

Force - ~~Dynes~~ Newton (10^5 dynes)

Velocity - Meter/Sec.

Acceleration - Meter/Sec.²

$$F = M A$$

$$= \text{Kg.} \cdot \text{Meter} / \text{sec}^2.$$

$$C_{\text{imp.}} = \frac{\text{Distance}}{\text{Force}} = \frac{\text{Meters}}{\text{Newton}}$$

$$\frac{Q}{V} = \frac{Q}{V} \quad E = IR$$

Mass = Force x Acceleration
(Newton) (M/sec²)
(Newton) (M/sec)

$$\text{Resistance} = \frac{\text{Force}}{\text{Velocity}}$$

Distance - Charge
Velocity - Current
Acc. - Rate of ch. $\dot{I}$
Force - ~~for~~ Volt.

For Osc. of 1 C.P.S.

$$f = \frac{1}{2\pi\sqrt{LC}}$$

$$T = 2\pi\sqrt{LC} = 1 \quad T^2 = 2\pi LC$$

$$I = 4\pi^2 LC \approx 4.9 \cdot \text{Mass} \cdot \frac{\text{Distance}}{\text{Force}} \quad \left[\frac{\text{mass}}{\text{sec}^2} \right]$$

dimensions
O.K.!

$$\frac{1 \text{ sec}}{36 \times 70} = \frac{4.9 \times 70 \text{ kg} [\text{C mech}]}{2520} \approx .4 \times 10^{-3} \text{ Meters/newton}$$

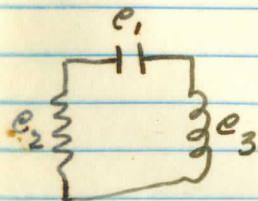
$$r^2 = \frac{M L}{M T^2}$$

$$\approx 1.7 \times 10^{-3} \text{ Meters} / \#$$

$$(N \cdot \cancel{M} / \text{sec}^2) \cdot \frac{M}{\underbrace{(N \cdot \cancel{M} / \text{sec}^2)}_n \underbrace{(M / \text{sec}^2)}_A}$$

CK Dimensions -

Analog



$$\frac{Q}{C} + \frac{dQ}{dt} R + \frac{d^2 Q}{dt^2} L = 0$$

$$\frac{X}{\tau_{cap}} + Res \frac{dx}{dt} + mass \cdot \frac{d^2x}{dt^2} = 0$$

$$\frac{M}{m/s} + \frac{N}{m/s} + N \cdot \frac{m}{s^2}$$

$$C_1 = \frac{Q}{C}$$

$$C_2 = IR = 49R$$

$$e_3 = d_1 L = L \frac{d_1^2}{d_2^2}$$

Equivalents:

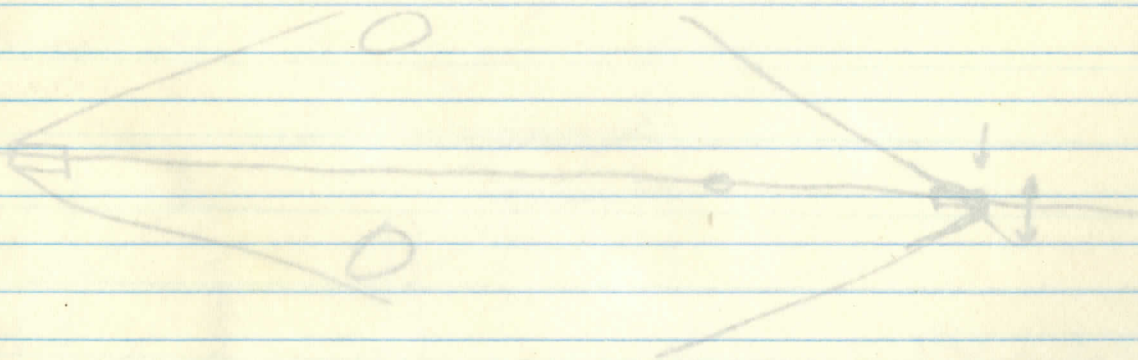
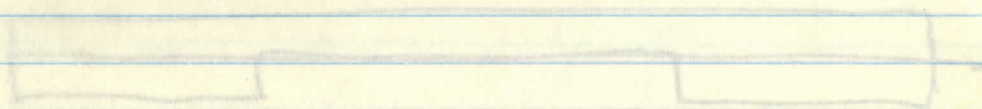
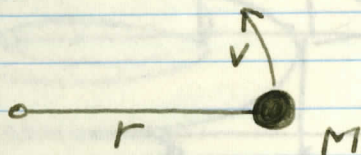
1 Newton = 10^5 Dynes = 3.60 oz

1# = 4.448 Newtons

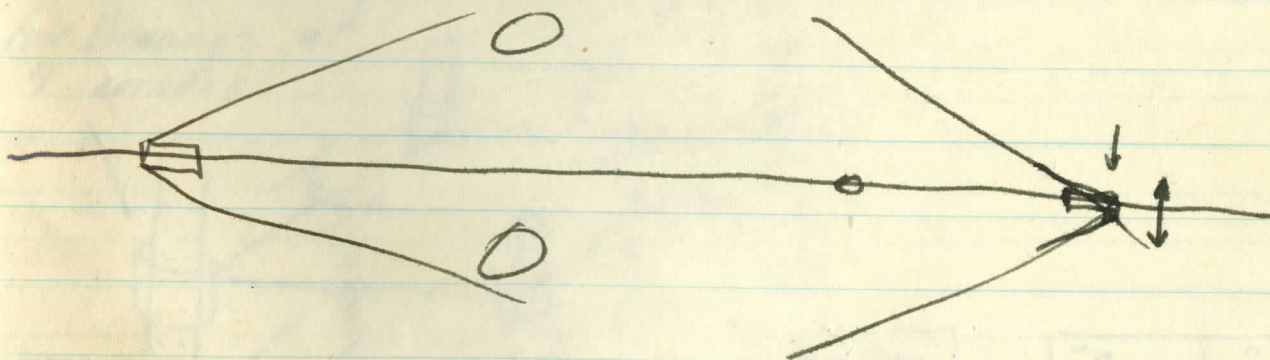
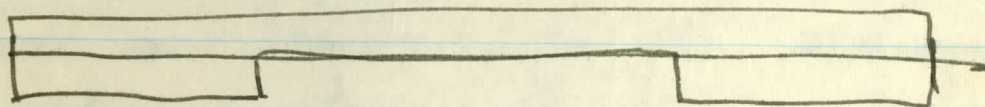
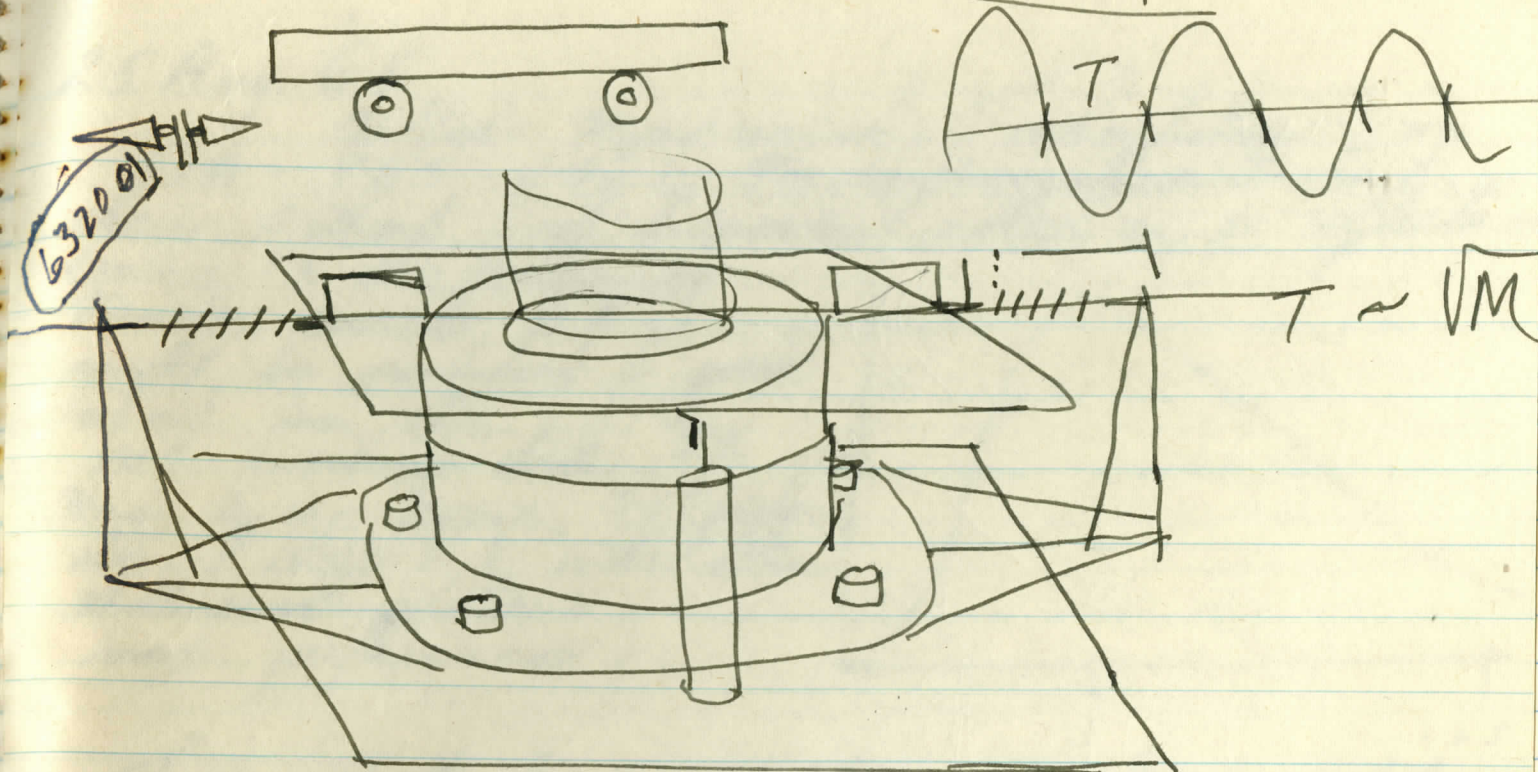
1 gm. wt. = 980 dynes

(Col. Carter)

Effects of a rotating wt.



(Col. Carter)

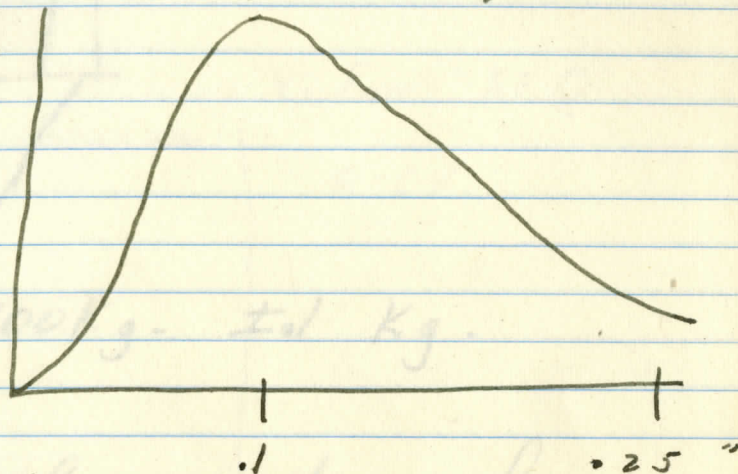


Counter
Type 1000
for 10

locked into posn. when not in use.

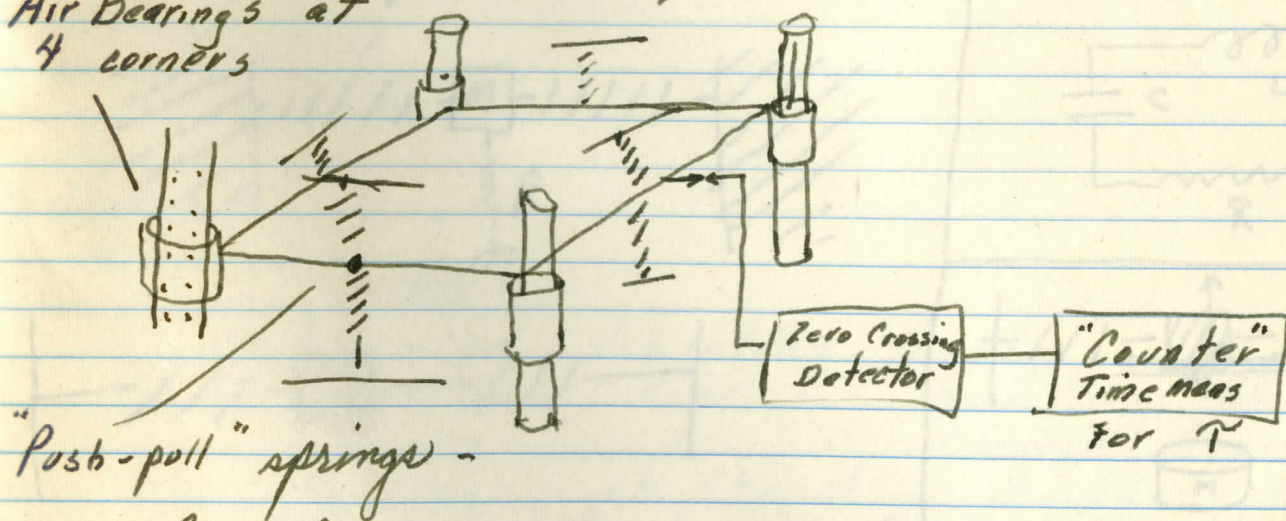
25 Aug. '65

Called Mechanical Technology Inc.
518 - 785 - 0922 Mr. Springstead about
their Optical Dist. Sensors which is a reflectance
device. I have the following
transfer curve but
might be modified to yield
 $\frac{1}{2}$ mil res. at
 $\frac{1}{4}$ " working dist. 2v.
Has 2v m.v. noise, $\frac{1}{4}$ " probe.
std. - 1000 + 2 wks. deliv.
It'll send quote +
max. performance -

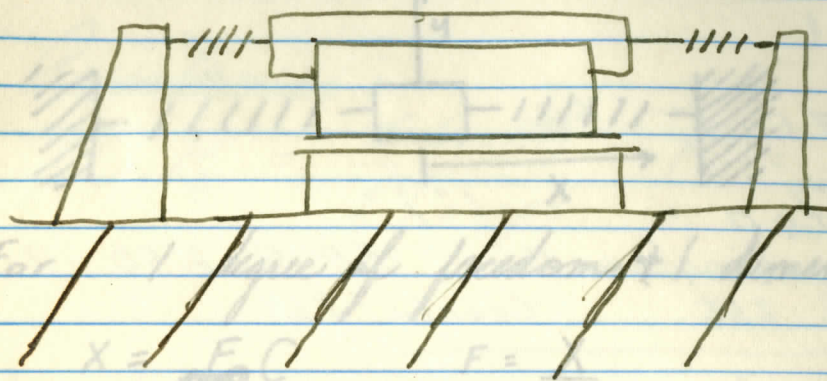


I'll do a simple exploratory model
using the air bearing as soon as possible.
Can see no reason why the scheme below
will not work in flight (weightless)

Air Bearings at
4 corners



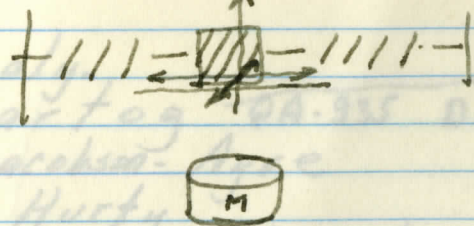
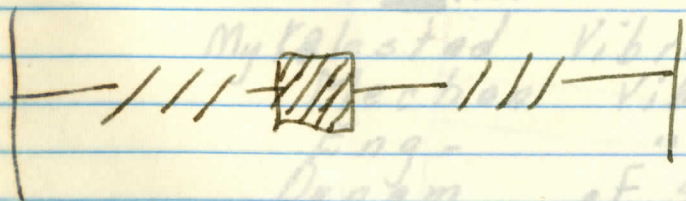
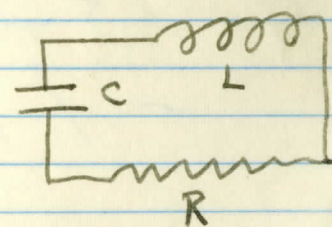
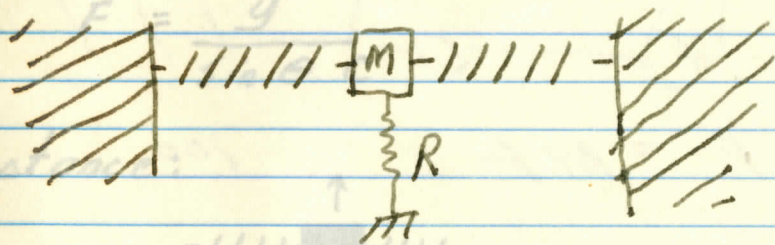
Such an arrangement could be incorporated
into a normal seat which would be
locked into posn. when not in use.



Range of Mass - 50-100Kg - ± 1 Kg.

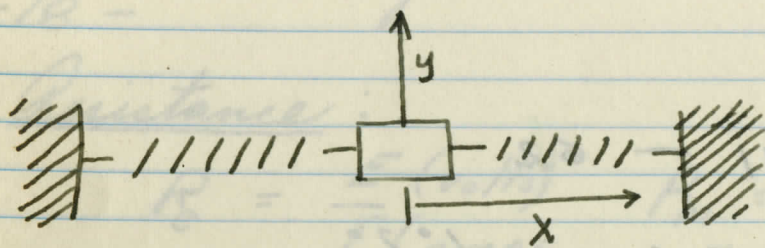
Accuracy 0.1%

Method: determination of period $\bar{t}$ the unknown mass made a portion of ~~the~~ an oscillating system at a nominal frequency of 1 C.P.S.



Quigly · Hughes

→ Earl Bell ←



For 1 degree of freedom + 1 dimension:

V: Force $X: Q$

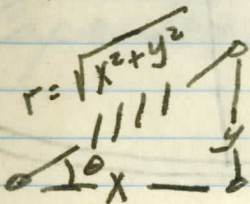
$$Q = CV$$

$$X = FC$$

$$F = \frac{X}{C}$$

$$F = \dot{X} R$$

For 2 degrees: dimensions:
Capacitance:



$$\sin \theta = \frac{y}{r}$$

$$\cos \theta = \frac{x}{r}$$

$$y = F \sin \theta C$$

$$F = \frac{y}{\sin \theta C}$$

Inductance:



Mykelestad Vibration - Analy.

Mechan Vibr. - Hartog QA-935 D3

Eng. " - Jacobson-Ayre

Dynam of Struct - Hurty

Vibr. Prob. in Eng. - Pambleschenko

RC-1100 S-72

Coernmann

QA-871 C-72

QA 871 KB2

RC-1125 H-27

TA-355 M74

TA-355 H27

QA-935 T4

Nortronics -

213 · 772 · 2321

Hughes -

213 · 391 · 0711 - 62

Paul Crafton · 1° 2°

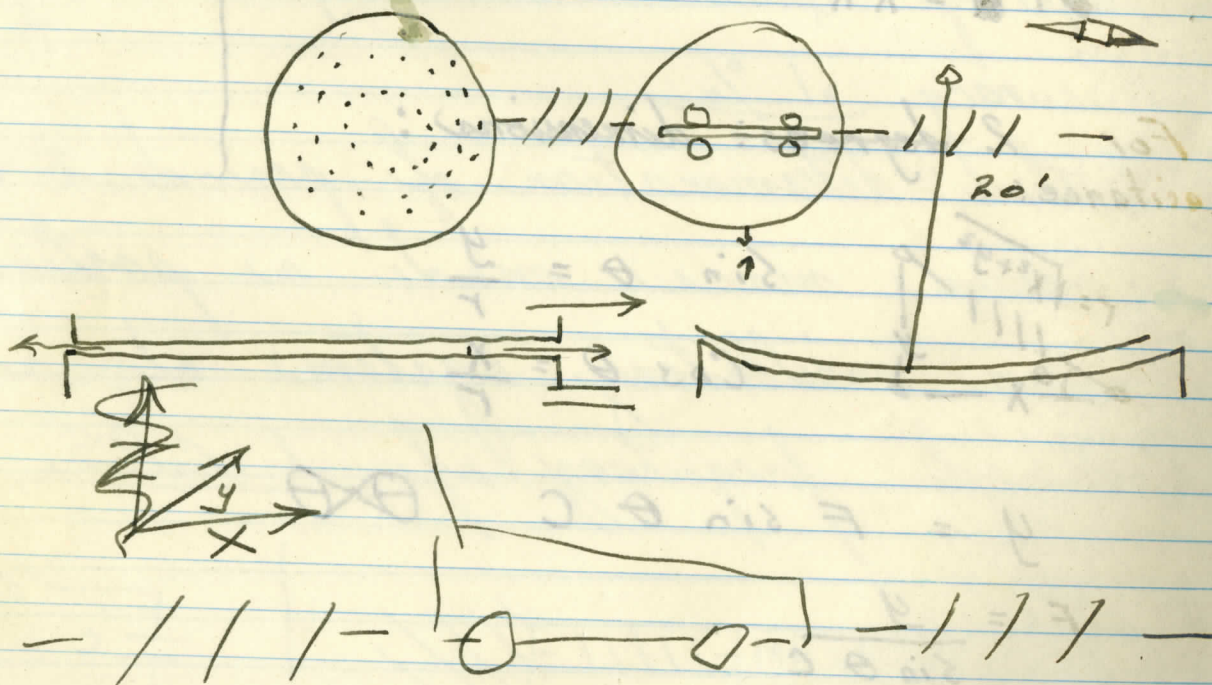
Geo. Washington Univ. FE 80250 · 202

Ext. 258

Cronin · 721

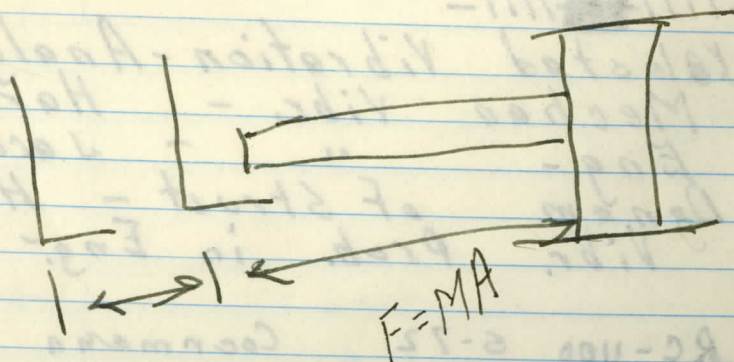
Dept. Mech.

Sgt.



$$F = MA$$

100



From Olson, Dynamical Analogies:

P-18 -

Resistance:

$$R_e = \frac{E \text{ (volts)}}{I \text{ (amps)}}$$

$$R_m = \frac{F \text{ (dynes)}_{\text{cgs}}}{\dot{x} \text{ (cm/sec)}_{\text{cgs}}}$$

Inductance:

$$E_{\text{(volts)}} = L_e \frac{dI}{dt} \left(\frac{\text{Amp}}{\text{sec}} \right)$$

$$F \text{ (dynes)} = M \text{ (grams)} \frac{d^2x}{dt^2} \left(\frac{\text{cm}}{\text{sec}^2} \right)$$

$$\& L_e = E \left(\right)$$

Capacitance