

Mass -

Exp -
#2

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Under conditions of space the physical
requirements for the experiment are met
by this system.

The practical arrangement of the
measuring device under conditions of
weightlessness is relatively simple.

The essential components are a
structure ^{to hold} the mass to be weighed,
^{spring assembly} ~~a~~ ^(springs) the mechanism for restoring forces, a
device to determine precisely when the
oscillating mass crosses zero point of

zero displacement, ~~and possibly of~~
a counter for determining time periods
calibrated in terms of mass and
some means of ~~AD~~ providing a displacement
along a given axis. ^{Fig -} The demonstration
device on earth is complicated by
the need for a frictionless support.

Considerations in determining
mass of the human body:

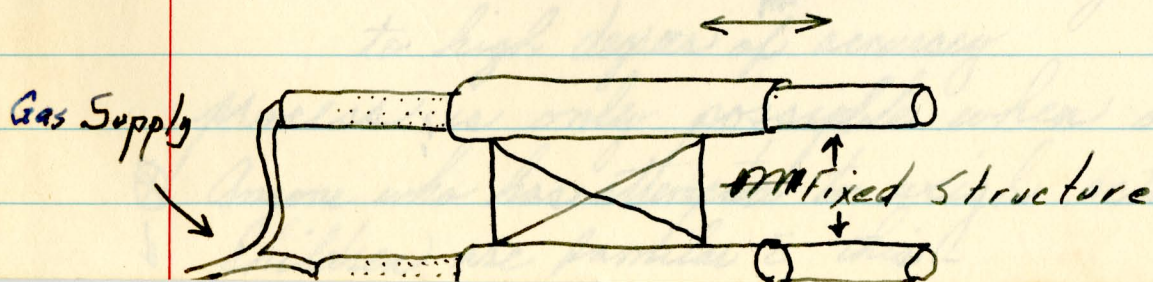
The body is composed of a number
of ~~or~~ flexibly connected segments

~~that~~ - such that determination of
the center of mass is only of academic
interest and could be expected to remain
constant for only brief periods. ~~App~~
~~argument which the first~~. Almost
as a corollary of this it becomes necessary
to restrict the motion to the minimum
degrees of freedom, which is rectilinear
motion in one pt. axis. Once this
is done it is immaterial where forces
and measurements are made so long

as they are in the parallel to the plane of motion. The practical details of this will be discussed further under the proposed demonstration.

One of the simpler ways of restraining motion to a single axis & minimum friction is the use of air bearings.

In this case the minimum ^{probably} required would be two cylindrical units as shown in -



The only ^{movements} shifts that would cause ~~in~~ an error in measurement would then be the components along the axis of oscillation and the error would be present only during the period of movement. ~~Now~~

~~in the same case exists in~~

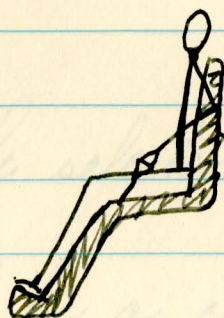
~~Now~~ This case is then the same as exists with an ordinary weighing

scale on earth. The weighing to high degrees of accuracy process is only possible when one is

⊗ Anyone who has attempted to weigh active children are familiar with this

standing or sitting quietly. ^{*} but when
As an aid in simulating ^{quiet standing posture} ~~this~~, and
also to provide ~~side~~ rigidity in an
oscillating system, some means should
be provided for rigid connection of the
subject to the ~~the~~ spring system as
well as ^{for} tensing the large muscles
of the body ^{to increase rigidity}. - This may be done
in the case of standing by simple
hand holds connecting to a foot plate.
See F - 
In the case of sitting both

a foot board correctly located to allow pressure against the back and also hand holds as in Fig — — —



① should accomplish this

The internal viscera are quite free to move but from studies done it appears that below certain critical accelerations, the viscosity + resistance of these organs are sufficient to cause the body to move as a

whole. But Walter had of vaccination
But not to my knowledge then established
D. certainty.

In any event the selection of frequency and
amplitudes must be a compromise between

A. ^{short} shortest possible measurement time
(this may later be shown to be not so
important a consideration)

B. Smallest displacement consistent $\bar{c}$
^{position}
accurate ~~amplitude determination~~ resolution

At amplitudes of 21 in. peak to
peak and

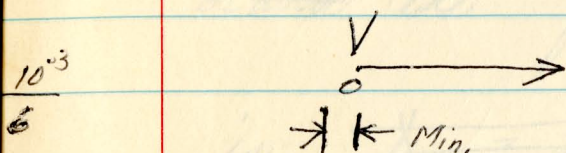
c- lowest frequency consistent c both possible as a function of A.

If the max. & min. G required to produce relative body motion were known ^{this} it would be ~~possible~~ to set the upper limits on freq-amplitude. In the
Without this knowledge one must make estimates and be prepared to alter these estimates in view of experimental results.

A frequency of 1 C.P.S. at

The displacement however has a direct bearing on accuracy since

it is the resolution of distance which ultimately determines the time resolution as follows. $\Delta T = \frac{\text{Min. dist. resolution}}{\text{Velocity at time of measurement}}$



Thus at the point of crossing zero displacement, the only logical point independent of amplitude changes, the relationship is as follows.

$$\Delta T = \frac{\text{Dist. Resolution}}{\text{Velocity} \cdot X_0 \cdot 2\pi f \cos \omega t}$$

$$\text{at } \omega t = 0 \text{ for } = \frac{\text{Dist. Resolution}}{X_0 \cdot 2\pi f}$$

For a min. time resolution

of $\frac{1}{2} \times 10^{-2}$ m.s. ($\frac{1}{2}$ inch) and a distance resolution of 10^{-3} " (.01")

the product $X_0 \cdot 2\pi f = 1$

and the frequency displacement relationship

is ~~$X_0 = \frac{1}{2}$~~ $f = \frac{1}{2\pi X_0}$.

If an amplitude of $\frac{1}{2}$ " is allowed the ~~max.~~ ^{min.} freq - allowed is $f = \frac{1}{2\pi \frac{1}{2}} \approx \frac{1}{\pi}$ c.p.s.

~~Acceleration (peak)~~

~~Peak acceleration occurs at $\frac{1}{2}$~~

Acceleration is given by

$$\frac{d^2 X}{dt^2} = -X_0 (2\pi f)^2 \sin \omega t$$

and peak acceleration occurs when
 $\sin \omega t = \pm 1$. In terms of G units

$$\text{Acceleration}_{(\max)} = -X_0 (2\pi f)^2 \cdot .0026 \cdot \pm 1$$

X_0 = initial displ. (ins)

f = (cycles/sec)

~~Acc = G on~~

For $X_0 = 1''$ and $G_{\max} = 10^{-2}$

$$\sqrt{\frac{10^{-2}}{2.6 \times 10^{-3}}} = 2\pi f$$

$$\& f = \frac{1}{2\pi} \sqrt{\frac{1}{2.6} \times 10}$$

$$\approx \frac{1}{2\pi} \sqrt{\frac{10}{2.6}} \approx \frac{1}{6} \sqrt{10}$$

$$\approx 1 \text{ cps}$$

CK

8
 3.08
 26) 1.00
 98
 200
 220
 208

$\frac{1}{6}$

From the foregoing it seems reasonable
to operate in the region of
a frequency of $\frac{1}{3}$ CPS and
displacements of 0.5 - 1.0 "

Many other factors will be present.
but most are not amenable to mathem

Another ~~factor~~ factor is
~~the relationship of effort~~ that the
space ship is a finite mass is present in the
space ship - The equation for
natural frequency under this
and the space ship will undergo

small displacements itself which will alter the frequency as follows -

$$\begin{array}{c}
 \boxed{M_1} \\
 \text{---} \\
 K \text{ ---} \text{---} \text{---} \text{---} \text{---} \\
 \text{|||||} \\
 \text{Rigid attachment}
 \end{array}
 \quad
 \begin{array}{c}
 \boxed{M_2} \\
 \text{---} \\
 \text{---} K \\
 \text{---} \\
 \boxed{M_2}
 \end{array}
 \quad
 f = \frac{1}{2\pi} \sqrt{K \left(\frac{m_1 + m_2}{m_1 \times m_2} \right)}$$

Thus for a normalized freq. (say

(C.R.S.) the change will be $\bar{c} \quad m_1 = 200 \#$

$m_2 = 39000 \#$

$$\frac{2 \times 10^2 + 3 \times 10^4}{2 \times 10^2 \times 3 \times 10^4}$$

$$\frac{3.02 \times 10^4}{6 \times 10^6}$$

$$\approx \sqrt{\frac{1}{2} \times 10^{-2}}$$

$$\approx \sqrt{50} \cdot \sqrt{10^{-4}}$$

$$\approx 7 \cdot 10^{-2}$$

$$\approx 7\%$$

The measurement will then yield a ^{slightly} higher frequency which will be ~~related to the true~~ result in ~~an apparently a decreased~~ the mass being read short i.e. less than actual mass.

For example:

$$\text{Mass (measured)} = \frac{\text{Mass (True)} \cdot \text{Mass (ship)}}{\text{Mass (True)} + \text{Mass (ship)}}$$

$$\text{Mass (True)} = 200 \text{ #/g}$$

$$\text{Mass (ship)} = 30,000 \text{ #/g}$$

$$= \frac{200 \times 30,000}{200 + 30,000}$$

$$= \frac{6 \times 10^6}{3002 \times 10^4}$$

$$= 199.866$$

or the measurement will be in
error by $\approx -0.13^\circ$ in 200° .

Since the ^{ships} mass of remains essentially
constant, this error will also remain
constant. It is to be assumed that
the mass system will be calibrated in
absolute terms when on station -

Another area of possible concern
are the characteristics of the springs -

• If one purchases ~~sp~~ precision
spring scale units, this should

case to be a problem. Some characteristics of an premium

150-^{Elastic} Alloy spring are given*

Thermal coeff. ^{of modulus} ϕ - -20 to $+15 \times 10^{-6}/^{\circ}\text{F}$
(this coeff. is apparently variable to account for other temp. errors in ~~spring~~ scale mechanisms)
& may theoretically be made zero

Hysteresis error - $\leq .05\%$ of deflection
(this will be equivalent to a damping resistance)

Creep error - $\leq .02\%$ of deflection / 5 min
~~(not applicable in oscillate)~~

If the mechanism is properly unloaded when not in use the characteristics

are beyond our required accuracy & stability -
* taken from John Chatillon & Sons
bulletin 1-SD-1

A number of 2 or 3rd order effects
may be present but do not
lend themselves to computation
but may be determined by measurement.

Finally there is the problem of
a realistic demonstration ^{under gravity}. After discussion

The Lockheed demonstration was,
as it has been misunderstood, unfortunate.

Mr. Riley It was only intended
to indicate methods & some
feasibility & using very crude

techniques. The remarkable point is that accuracies were as high as they were - Mr. Riley (proj. engineer) ~~states~~ has no reservations about meeting the ± 0.1 pound accuracies & equipment designed for this accuracy.

However the problem is now to make a convincing demonstration on earth. The earth demonstration will be more difficult than under free fall since the ~~gravitational~~ some

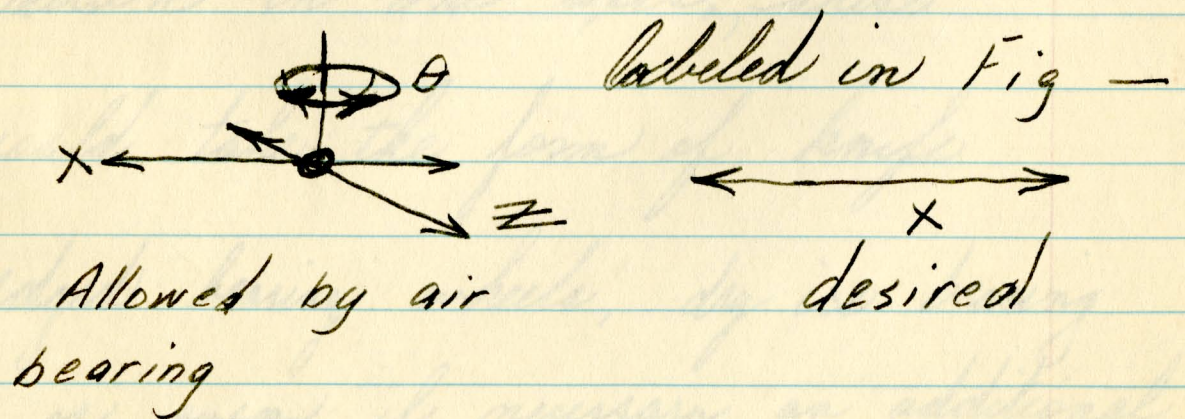
* The bearing is a 12" circular unit constructed by Astro Physics Labs, Huntsville for a ballistocardiograph. According to its designers + fabricators it is capable of ~~±~~ $\pm .75"$ of motion ~~& resistance~~ resistive forces of ~~a few~~ 1-2 dynes. It has a circular ground face on a radius of 20 ft. which will provide a restoring force toward center which will be a function of angular displacement and weight. This restoring force will be analogous to an additional spring force but will not be considered here since it should remain a system constant.

must be present
form of bearing, to offset gravity
3 introducing unwanted frictional effects.

Thus in addition to the items listed
on page — must be added this
bearing as an essential component
of the system on earth. Of all
available bearings an air bearing
is most nearly the ideal & ~~of~~
negligible friction coefficients. Now

Since such a bearing is
available the demonstration is built
around this unit for expediency. *

~~The bearing itself~~
 Unfortunately this bearing is
 a circular unit which ~~is~~ ^{translatory} additional
 restraints will allow ^{motion}
 in an additional axis ~~as~~ and
 rotatory motion will allow
 rectilinear motion in two axes
 and rotatory motion about another
 axis. These motions have been



labelled. Two of the degrees of freedom of motion, in the x axis & Θ rotational plane are undesired. The bearing is being used to simply support the weight and because of its availability. It may be necessary to add restraints to contain the motion in one axis; these could take the form of knife edged bearing wheels, dry ice bearing or even if necessary an additional